

The problem asks for the induced thermal stress of a solid steel shafting subjected to a temperature rise from **35°** to **120°**.

**If a body is free to expand or contract, no stress is induced. If expansion or contraction is restricted, thermal stresses develop."**

The problem statement fails to explicitly state that the shaft is completely restricted or fixed between two rigid walls. If the shafting were simply lying freely on a shop floor, the induced thermal stress would be **0 MPa**. Furthermore, standard nominal properties for structural carbon steel must be pulled from the provided reference material constraints: Modulus of Elasticity (E) is taken as 200GPa ( $200 \times 10^3$  MPa) and the coefficient of linear thermal expansion ( $\alpha$ ) for steel is exactly  $10.8 \times 10^{-6}$  m/m-°C.

### **Step 1: Geometric Expansion and Thermal Strain Formulation**

When a material experiences a temperature shift, its internal kinetic energy increases, forcing the atomic lattice structure to expand. The unconstrained thermal deformation ( $\delta_{th}$ ) is directly proportional to its initial length ( $L_o$ ), the temperature differential ( $\Delta T$ ), and its material-specific expansion rate ( $\alpha$ ).

Mathematically, this unhindered elongation is expressed as:

$$\delta_{th} = \alpha L_o (t_2 - t_1)$$

However, when rigid limits prevent this physical movement, the walls exert an equal and opposite mechanical reaction force (F) against the expanding shaft. This force creates a compressive mechanical deformation ( $\delta_{mech}$ ) that completely neutralizes the thermal expansion.

According to Hooke's Law, this axial mechanical deformation is calculated as:

$$\delta_{mech} = \frac{FL_o}{AE}$$

Because the total net physical change in length equals zero ( $\delta_{net} = \delta_{th} - \delta_{mech} = 0$ ), we set the two expressions equal to each other:

$$\alpha L_o (t_2 - t_1) = \frac{FL_o}{AE}$$

### **Step 2: Isolating for Induced Stress (S)**

Notice that the initial length ( $L_o$ ) appears on both sides of our equilibrium equation. Dividing both sides by  $L_o$  eliminates the length parameter from the equation entirely. This is a vital conceptual realization for board exam questions: **the induced thermal stress in a completely restricted, uniform member is entirely independent of its length and cross-sectional area.**

Recall that normal stress (S) is defined as force per unit area ( $S = \frac{F}{A}$ ) Substituting S into the remaining terms yields the classic thermal stress equation:

$$\alpha(t_2 - t_1) = \frac{S}{E}$$

Isolating the induced stress (S), we get:

$$S = \alpha E(t_2 - t_1)$$

### **Step 3: Numerical Substitution and Calculation**

We can now plug our values into the isolated thermal stress formula.

- $\alpha = 10.8 \times 10^{-6} / ^\circ\text{C}$
- $E = 200 \text{ GPa} = 200,000 \text{ MPa}$
- $t_1 = 35^\circ\text{C}$
- $t_2 = 120^\circ\text{C}$

$$S = (10.8 \times 10^{-6} / ^\circ\text{C})(200,000 \text{ MPa})(120 - 35) ^\circ\text{C}$$

$$S = 183.6 \text{ MPa}$$

The most profound lesson to take away from this problem is the invisible, immense power of thermal energy. By simply heating this stationary, fixed piece of steel by 85°C, a massive internal compressive stress of **183.6 MPa** is generated. For a 6 cm diameter shaft, this translates into a crushing physical force of over **524 kN** (roughly 118,000 lbs of force) pushing outward against the boundaries.